

[54] **ROLAMITE SAFETY AND ARMING MECHANISM**

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[51] Int. Cl. .... F42c 15/24

[58] Field of Search..... 102/78, 70, 70.2, 79, 80

[56] **References Cited**

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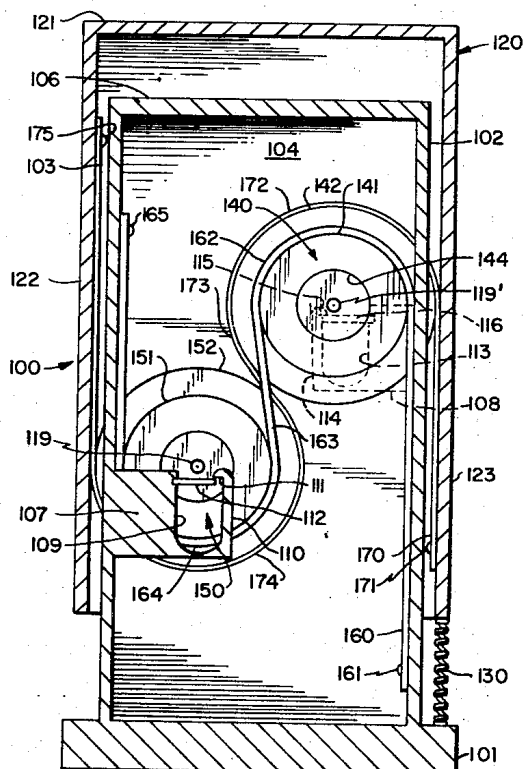
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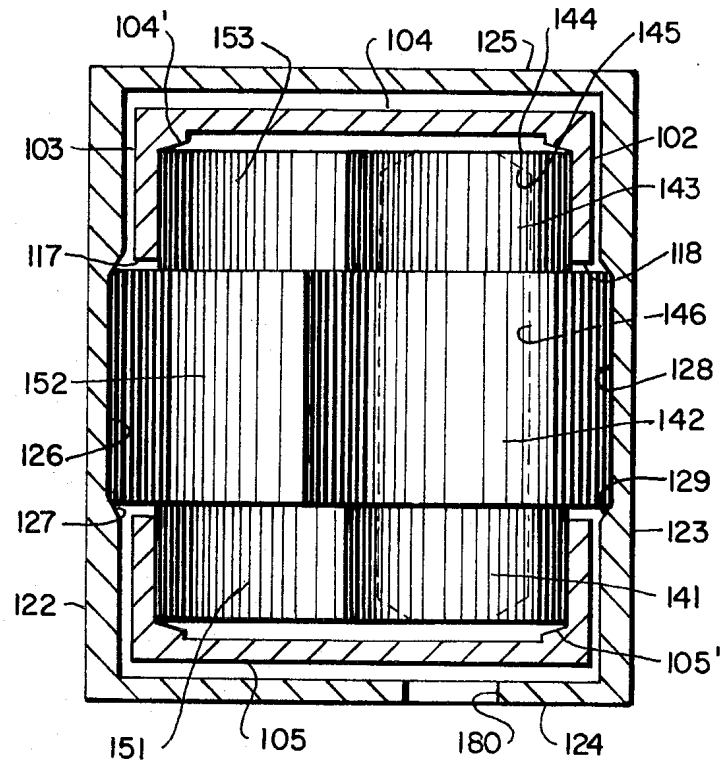
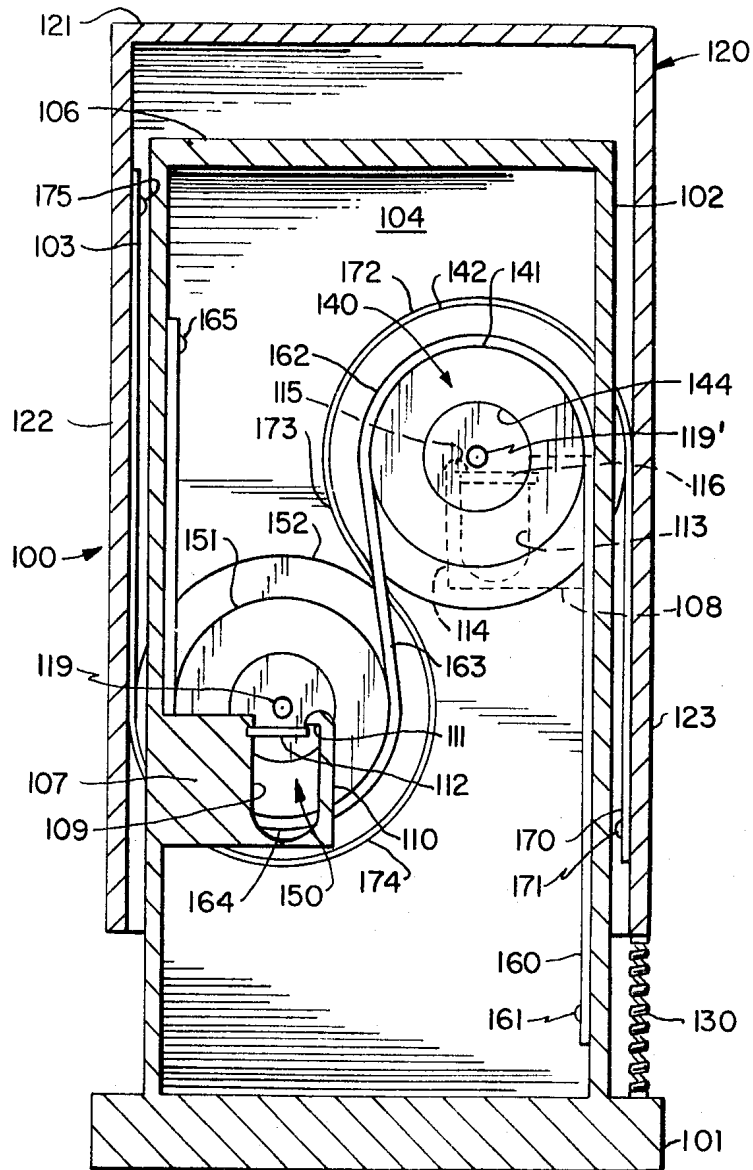
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[57] **ABSTRACT**

An inertial operated fuzing and arming mechanism having two telescoping encasing members and a pair of rollers held within said encasing members and a pair of rollers held within said encasing members by a plurality of S-shaped flexible bands, each band engaging different diameter portions of the rollers and the ends of the bands being attached to the inside of said encasing members. A detonator and explosive are associated with one of said encasing members and one of the rollers and the other encasing member has an aperture therein so that when said two encasing members and said roller are aligned, the fuze is activated to detonate the explosive.

**11 Claims, 4 Drawing Figures**





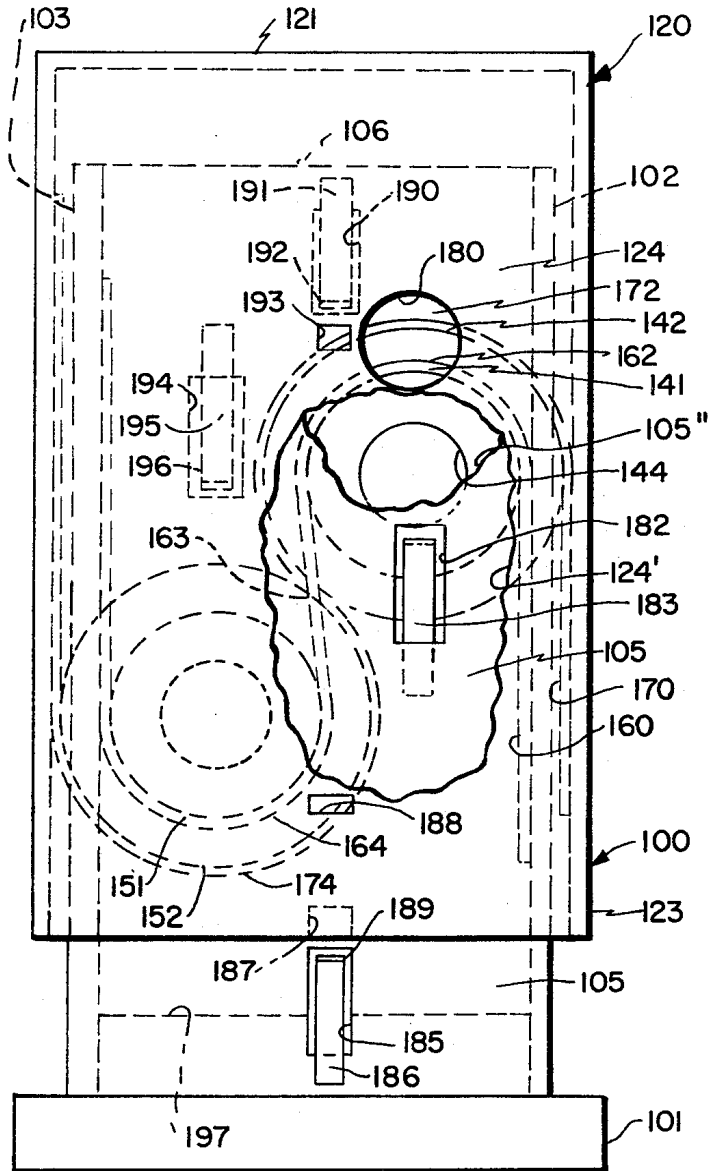
**FIG. 1**

**FIG. 3**

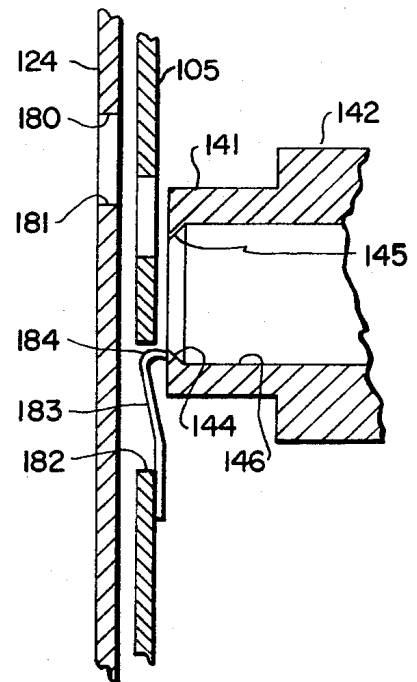
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**FIG. 2**



**FIG. 4**

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**ROLAMITE SAFETY AND ARMING MECHANISM**

The invention described herein may be manufactured, used and licensed by or for the United States Government for governmental purposes without the payment to me of any royalty thereon.

This invention relates to a new fuzing and arming mechanism which utilizes the roller-band method of energy transfer to arm a fuze.

Prior art fuzes have employed various means and methods to arm fuzes used to detonate explosives in projectiles, canopy releases and similar applications. These means have included accelerometers and G-switches which have been limited in their accuracy, sensitivity and efficiency by the degree of sliding friction inherent in these devices or due to structural inaccuracies in the elements used in the device.

Attempts to increase their sensitivity by decreasing sliding friction in accelerometers tends to decrease the force applied between the electrical contacts closed by the mechanisms reaction to an acceleration force. These mechanisms generally may not be adjusted to accurately respond to a relatively wide range of acceleration forces due to the unpredictable and variable frictional losses between the components of the mechanism.

Other attempted solutions to minimize sliding friction involve lubricants, both gaseous and liquid. Such lubricants can create forces detrimental to the operation of certain mechanisms under highly sensitive operating conditions. There is always the danger of contamination of critical areas within the housing thus creating the need for packing and sealing means.

The use of bearings has been limited by the manufacturing problems in fabricating precision surfaces and attaining close tolerances. Resulting misalignment and rough surfaces minimize overall efficiency.

The present device is only involved with rolling friction, the coefficients of which are considerably lower than a sliding coefficient of friction. Due to the low coefficients of friction, it is possible to construct the instant mechanism according to microminiaturization techniques.

Accordingly, it is an object of this invention to provide a fuzing and arming mechanism having low friction losses.

It is a further object of this invention to provide a fuzing and arming mechanism having only rolling friction losses.

It is a further object of this invention to provide a fuze mechanism for employing two telescoped members within which a pair of rollers are encased for movement therein.

It is a further object of this invention to provide a fuzing and arming mechanism which will not operate until a bias acceleration has been achieved.

These and other objects of the invention will become apparent when taken with reference to the following drawings in which:

FIG. 1 is a sectional side elevation view of the fuzing and arming mechanism showing one type of safety latch;

FIG. 2 is a side elevational view of the fuzing and arming mechanism similar to FIG. 1 but showing the aligning aperture in the case and a different type of safety latch and showing portions of the case and housing broken away.

FIG. 3 is a sectional view of the portion of FIG. 2 which is broken away only the housing and case walls are shown in full to show the relationship of the alignment of the rollers, case and housing;

FIG. 4 is a plan sectional view of the mechanism of FIGS. 1 and 3 showing the relationship of the rollers to the case and housing.

Portions of the mechanism which constitute this invention function according to the principles described in U.S. Pat. Nos. 3,452,175 and 3,471,668. The instant device is different from the devices described in those patents since there are two separate band systems acting on the rollers and each band system is connected to a separate encasing member, the members being movable relative to each other.

Referring now to FIG. 1, there is shown the preferred form of this invention, generally designated as 100. It consists of a

flat base member 101 having a generally rectangular housing affixed thereto. The housing consists of walls 102, 103, 104 and 105 (FIGS. 2 and 3) and a cap portion 106.

Affixed to wall 103 is a projection 107 having a latch 112 of resilient material which is carried by an extension 110 of projection 107. The projection 110, latch 112 and projection 107 form a U-shaped area 109. Hook portion 111 on extension 110 prevents latch member from being bent outwardly and only allows it to be bent inwardly in relation to area 109.

A second projection 108, identical to projection 107, has an extension 114 which supports a resilient latch 116 which is prevented from being bent upwardly by hook portion 115. Latch 116, extension 114 and projection 108 form a U-shaped area 113. Projection 108 and extension 114 are secured to wall 105. Areas 109 and 113 are adapted to receive latch pins such as 119.

Wall 103 of the housing has a vertical slot 117 therein and wall 102 has a similar slot 118 therein (FIG. 3).

Surrounding the housing is a case generally designated as 120. It has a top portion 121 and side walls 122, 123, 124 and 125.

The case is generally rectangular when viewed from the top (FIG. 3) and has slightly larger dimensions than housing 100 to enable it to fit over the housing and vertically slide relative thereto. A spring 130, as shown in FIG. 1, may connect the case with the housing.

Walls 122 and 123 of the case 120 having vertically extending relieved areas 126 and 128 which are beveled as at 127 and 129, respectively. Areas 126 and 128 are aligned with slots 117 and 118, respectively, in housing 100.

Mounted inside of housing 100 are a pair of rollers 140 and 150. Roller 140 has two external cylindrical portions 141 and 143, of equal diameter, and a central cylindrical portion 142 of a diameter larger than that of portions 141 and 143.

A central cylindrical bore 146 (FIG. 3) in roller 140 terminates at each end thereof in necked surfaces such as 145 and openings 144. The internal bore may carry either a detonator or an explosive material.

Roller 150 is identical with roller 140 and has portions 151, 152 and 153 which correspond with portions 141, 142 and 143 on roller 140. Extending from the roller 150, which may carry detonator or explosive in a bore similar to bore 146 is a latch pin 119 which is adapted, as roller 150 moves downwardly in FIG. 1, to bias resilient latch member 112 open and enter area 109. Similarly, a latch pin 119' on roller 140 can enter area 113. The resilient latch members 112 and 116 interact with hook portions 111 and 115 to prevent movement of the rollers other than that occurring from the travel of latch pins 119 and 119' within areas 109 and 113.

Securing rollers 140 and 150 in place are three bands although two may be used. A first band 160 is secured at one end thereof to the inner surface of wall 102 by any suitable means such as screw 161. Band 160 extends upwardly along wall 102 and around cylindrical portion 141 of roller 140 as at 162 down as at 163 and around cylindrical surface 151 of roller 150 as at 164 and up along the inner surface of wall 123, around the central cylindrical portion 142 of roller 140 as at 172 which projects outwardly through the elongated vertical slot 118 in wall 102 of housing 100 down as at 173, around the central cylindrical portion 152 of roller 150 as at 174 which projects outwardly through the elongated vertical slot 117 in wall 103 of housing 100 and up along the recessed area 126 of the inner surface of wall 122 to a point of attachment to the case such as screw 175.

It is obvious that a third band may be used which would be identical with band 160 and would be in contact with the housing and cylindrical surfaces 143 and 153 of the rollers 140 and 150.

Wall 124 has an aperture 180 therein by which the explosive and detonator (one of which is carried externally of the case and the other of which is carried by either of the rollers) can align when the mechanism is subjected to a predetermined acceleration in the vertical direction. Obviously, there is a

second aperture (not shown) provided in wall 105 of housing 100.

The operation of the mechanism comprising the rollers, bands, housing and case is initiated by a positive or negative acceleration in the vertical direction. Any component of acceleration in the vertical direction can actuate the mechanism. Either a linear or centripetal acceleration or a combination of the both will be detected by the mechanism and if the magnitude and duration of the accelerating environment is sufficient, the mechanism will operate and can be used to arm a fuze.

With a vertical component of acceleration the mechanism can be designed such that the case will move downwardly with respect to the housing and the rollers will move upwardly relative to the housing so that the bore of roller 140, the aperture (not shown) in wall 105 and the aperture 180 in wall 124 will align. When such an alignment occurs, the rollers, housing and case can be held in that position by the proper positioning of the latches within housing 100.

As an alternative, the mechanism can operate in a reverse manner such that when the housing 100 is subject to an acceleration in the upward vertical direction, the case 120 will accelerate relative to the housing in the upward vertical direction and the rollers will accelerate, relative to housing 100, in the downward vertical direction. This, of course, is achieved by the direction in which the bands are secured within the case and housing, the tension of the bands and the force exerted by the spring 130.

When the holes in the case 120 and housing 100 are aligned with the bore of a roller, the detonation train is complete.

The displacement of the rollers relative to the housing does not have to be equal in magnitude to the displacement of the case with respect to the housing. Neither the rollers nor the case need have displacements relative to the housing that are large relative to the motion of the frame.

The band construction can be varied to build in a bias acceleration to the mechanism. This means that no relative motion between the housing, case and rollers will occur when the mechanism is subjected to an acceleration smaller than the bias acceleration. The bias acceleration can be built in by either cutting out portions of the band or by preforming the band or by doing both.

Referring now to FIG. 2, the mechanism of FIG. 1 is shown in elevation and incorporating a different type of latching means from that shown and described in FIG. 1.

The wall 124 of case 120 is shown broken away as at 124' to show wall 105 of housing 100 in FIG. 2. Wall 105 is shown broken away as at 105'' to show roller 140. FIG. 4 shows the relationship of these areas of walls 105 and 124 in relation to roller 140.

As seen in FIG. 2, a rectangular slot 182 is shown in wall 105. Mounted on the inner surface of wall 105 is a resilient latch 183 having, as seen in FIG. 4, a hooking portion 184 which is adapted to ride on the end surface of cylindrical portion 141 of roller 140.

The particular mechanism shown in FIGS. 2 and 4 is adapted to allow the rollers 140 and 150 to move upwardly upon acceleration in the upward vertical direction. However, the hooking portion 184 of latch 183 is adapted to enter the opening 144 of bore 146 in roller 140 and prevent further downward travel of roller 140 should a band break or the end of a band break loose from either the case 120 or housing 100.

As seen in FIG. 4, wall 105 of housing 100 has an aperture 181 which is adapted to align with opening 144 in roller 140 and aperture 180 in wall 124 of case 120.

To secure the case 120 to housing 100 in case of band breakage, a rectangular opening 185 is provided in housing wall 205 to allow biasing of a resilient latch 186. The end of latch 186 is attached by any suitable means to the outer surface of wall 105 and the hooking portion 189 of which is adapted to first slide on beveled area 187 on the inside of case wall 124 to bias the latch as the case slides down over the housing and then enter a rectangular locking aperture 188 in casing wall 124.

To insure against overtravel of rollers 140 and 150 in the upward direction, a rectangular aperture 194 is located on the upper housing wall 105 and allows for biasing of resilient latch 195 having a hooking portion 196. Portion 196 is adapted to bias inward to engage in the bore of roller 150 to prevent overtravel of the rollers in the upward position. In this sense, latch 195 acts in a manner similar to latch 183.

A rectangular opening 190 is located on the upper housing wall 105 and allows for biasing of a resilient latch 191 having a hooking portion 192. Portion 192 is adapted to swing outward to engage in aperture 193 in case wall 124 as the case 120 moves upwardly on housing 100.

The various latches can be used solely as safety latches, i.e., to prevent any overtravel of the rollers or case in relation to the housing to prevent damage due to band breakage or failure or they can be used to secure the various components in a position whereby the arming apertures, such as 180 and 181 are aligned with the bore of the roller carrying the detonator or explosive.

Both latches 186 and 195 swing inwardly to locking position and latches 186 and 191 swing outwardly to lock.

The bands of the mechanism may be formed to obtain constant force level positive and negative spring constants, second order and higher force curves and detenting actions, in any desired combinations. The bands are constructed of thin metals and may be fashioned so that their thickness and width vary along their length. They also may be cut out in areas to vary the spring modulus.

What is claimed is:

1. A safety and arming mechanism, said mechanism comprising a housing means, said housing means having a base and at least two vertically extending opposing walls, said walls having vertically extending slot means therein, a case means having a top and at least two depending opposing sides, said sides being separated by a distance greater than the space between said walls and said case means telescopically received over said housing means, a plurality of rotatable members intermediate said walls and movable vertically therein, said rotatable members having a first cylindrical portion of a given diameter and at least one second cylindrical portion of lesser diameter than said first surface, the cross section of said first portions having a combined cross section greater than the spacing between said sides and said second portions having a combined cross section greater than the spacing between said walls, said first cylindrical portions extending through said vertically extending slot means, a first flexible band looped in a substantially S-shaped configuration around said first portions and having each of its ends attached to said sides, respectively, a second flexible band looped in a substantially S-shaped configuration around said second portions and having each of its ends attached to said walls, respectively, said bands being under tension and allowing rolling movement of said members and relative movement between said housing and case upon attainment of a predetermined acceleration and fuze and explosive means associated with said housing, case and rotatable members and adapted to be activated when said housing, case and members attain a predetermined relative position caused by a predetermined acceleration.

2. A mechanism as in claim 1 wherein said rotatable members have a third cylindrical portion, said third portion being of equal diameter with said second portion, and a third flexible band looped around said third portions in a substantially S-shaped configuration and having each of its ends attached to said walls respectively.

3. A mechanism as in claim 2 wherein said second and third portions of each said rotatable member are coaxial with said first portion thereof and are on each side of said first portion.

4. A mechanism as in claim 1 including safety means to prevent movement of said members and arming of said fuze means upon breakage of any of said flexible bands.

5. A mechanism as in claim 4 wherein said safety means includes an opening in one of said housing walls, a resilient latch means protruding through said opening, one of said rotatable members having a receiving means adapted to coact with said

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resilient latch means to prevent accidental displacement of said rotatable member within said housing.

6. A mechanism as in claim 4 wherein said safety means includes a first opening in one of said housing walls, a resilient latch means protruding through said opening, a second opening in one of said depending sides vertically aligned with said first opening and adapted to receive said latch means upon accidental displacement of said case relative to said housing.

7. A mechanism as in claim 4 wherein said safety means includes a pin on at least one of said rotatable members, and extending coaxially therefrom, a pin receiving means on said housing, a resilient latch means associated with said pin receiving means and adapted to bias in only one direction so as to allow said pin to enter said receiving means but prevent it from exiting.

8. A mechanism as in claim 1 including a spring means connected to said housing and said case, said spring means along with the tensioned flexible bands determinative of the amount of acceleration necessary to initiate relative movement between said case and housing.

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9. A mechanism as in claim 1 wherein said fuze and explosive means includes a charge of explosive and a detonator means, one of said explosive and detonator means being located on said case and the other being associated with one of said rotatable members.

10. A mechanism as in claim 9 wherein one of said rotatable members has a bore therein, said explosive being located within said bore, a first aperture in said housing wall vertically aligned with said bore, a second aperture in the depending side of said case adjacent said housing wall having said first aperture and vertically aligned with said first bore, said detonator means adapted to act through said first and second apertures when they become horizontally aligned with each other and said bore to detonate said explosive upon said mechanism being acted upon by an acceleration.

11. A mechanism as in claim 9 wherein said detonator means is located within said bore and said explosive means is associated externally with said case means.

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